Helping a driver in backward docking with N-trailer vehicles by the passive control-assistance system

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- Introduction
- Vehicle kinematics and control objective
- Control-assistance for N-trailer vehicles
- Experimental results
- Conclusions

- 1 Introduction
- 2 Vehicle kinematics and control objective
- 3 Control-assistance for N-trailer vehicles
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Motivation

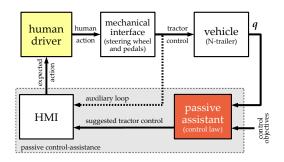
Introduction



- N-trailer vehicles highly nonlinear kinematics hard to control
- Maneuvers by-hand very difficult, nonintuitive and burdening (dangerous)
- Control-assistance for agile maneuvers especially desirable in this case



General concept of passive control-assistance

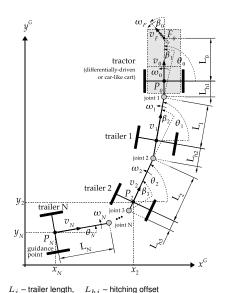


- q = measurable vehicle configuration
- Assistant suggests control action upon q and control objectives (human intuition NOT required)
- Control action performed entirely by a human driver (passive assistance ⇒ steer-by-wire and cruise control systems NOT needed)



- 1 Introduction
- Vehicle kinematics and control objective
- 3 Control-assistance for N-trailer vehicles
- Experimental results
- 6 Conclusions

Kinematics of N-trailer vehicle – definitions



Vehicle configuration vector

$$q \triangleq [\underbrace{\beta_1 \ldots \beta_N}_{q} \underbrace{\theta_N \ x_N \ y_N}_{q}]^{\top} \in \mathbb{R}^{N+3}$$
 (1)

Kinematic control input

$$\boldsymbol{u}_0 = [\omega_0 \ v_0]^{\top} \in \mathbb{R}^2 \quad \text{ for DDV} \qquad (2)$$

$$\boldsymbol{w}_0 = [\omega_F \ v_F]^{ op} \in \mathbb{R}^2 \quad \text{for CLV} \qquad \mbox{(3)}$$

DDV = Differentially Driven Vehicle
CLV = Car-Like Vehicle

Assumptions

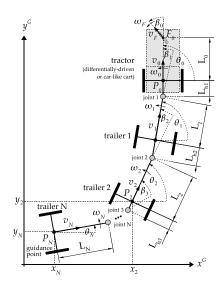
$$\begin{split} L_i > 0 \quad \forall \, i = 1, \dots, N \\ L_{hi} > 0 \quad \forall \, i = 1, \dots, N \text{ (nSNT)} \end{split}$$

nSNT = non-Standard N-Trailer

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Introduction

Kinematics of N-trailer vehicle - cascaded form



Kinematics of i-th segment

$$\dot{\theta}_i = \omega_i$$
 $\dot{x}_i = v_i \cos \theta_i$
 $\dot{y}_i = v_i \sin \theta_i$
 $\beta_i = \theta_{i-1} - \theta_i$

 ω_i , v_i – virtual inputs of *i*-th segment

Velocity transformation

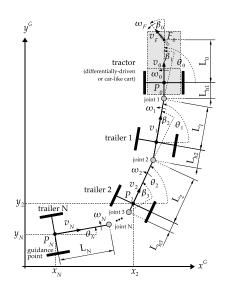
$$\underbrace{\begin{bmatrix} u_i \\ v_i \end{bmatrix}}_{\boldsymbol{u}_i} = \underbrace{\begin{bmatrix} -\frac{L_{hi}}{L_i}\cos\beta_i & \frac{1}{L_i}\sin\beta_i \\ L_{hi}\sin\beta_i & \cos\beta_i \end{bmatrix}}_{\boldsymbol{J}_i(\beta_i) \; \leftarrow \; \text{invertible for } L_{hi} \neq 0}_{\boldsymbol{u}_{i-1}} \underbrace{\begin{bmatrix} \omega_{i-1} \\ v_{i-1} \end{bmatrix}}_{\boldsymbol{u}_{i-1}}$$

Velocity propagation along a vehicle chain

$$egin{aligned} oldsymbol{u}_i &= \prod_{j=i}^1 oldsymbol{J}_j(eta_j) oldsymbol{u}_0, \quad oldsymbol{u}_{i-1} &= \prod_{j=i}^N oldsymbol{J}_j^{-1}(eta_j) oldsymbol{u}_N \ i &= 1, \dots, N \end{aligned}$$

Introduction

Control objective formulation (docking task)



Constant reference set-point

$$\boldsymbol{q}_{Nd} = [\theta_{Nd} \ x_{Nd} \ y_{Nd}]^{\top} \tag{4}$$

Weighted posture/docking error

$$oldsymbol{e}_w(t) riangleq oldsymbol{W} oldsymbol{e}(t), \quad oldsymbol{e}(t) = egin{bmatrix} e_{ heta} \\ e_{x} \\ e_{y} \end{bmatrix} riangleq oldsymbol{q}_{Nd} - oldsymbol{q}_{N}(t)$$

where $\boldsymbol{W} = \mathrm{diag}\{w,1,1\}$ and $w \in [0,1]$

Definition (Docking task)

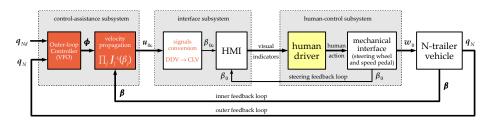
Compute control suggestions in the form of velocities $\boldsymbol{u}_0(\boldsymbol{e},\boldsymbol{\beta})$ and desired steering angle $\beta_{0c}(\boldsymbol{u}_0(\boldsymbol{e},\boldsymbol{\beta}))$ which guide the N-th trailer to set-point \boldsymbol{q}_{Nd} with prescribed accuracy $\delta>0$ guaranteeing that

$$\forall t \geq T \| \boldsymbol{e}_w(t) \| < \delta,$$

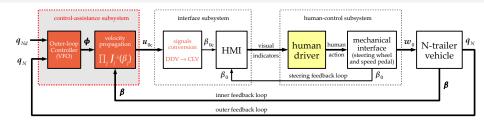
where $T \in (0, \infty)$ is a control time-horizon.

- 1 Introduction
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Cascaded structure of the motion control system



- N-trailer with CLV tractor \Rightarrow control input $w_0 = [\omega_F \ v_F]^{\top}$
- Human driver as a feedback controller of steering-angle β_0 and governor of driving speed v_F

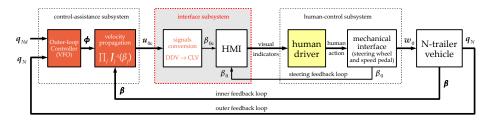


$$\boldsymbol{u}_{0c} \triangleq \left\{ \begin{array}{ccc} \prod_{j=1}^{N} \boldsymbol{J}_{j}^{-1}(\beta_{j}) \boldsymbol{\Phi}(\boldsymbol{e}) & \text{for} & \|\boldsymbol{e}_{w}\| > \delta \\ \boldsymbol{0} & \text{for} & \|\boldsymbol{e}_{w}\| \leq \delta \end{array} \right., \qquad \boldsymbol{\Phi}(\boldsymbol{e}) \triangleq \left[\begin{matrix} k_{a}(\theta_{Na} - \theta_{N}) + \dot{\theta}_{Na} \\ \|\bar{\boldsymbol{e}}\|^{\gamma} \cos \alpha \end{matrix} \right]$$

$$\begin{split} \parallel \bar{\boldsymbol{e}} \parallel &\triangleq \sqrt{e_x^2 + e_y^2} \\ h_x &\triangleq k_p e_x - \eta \sigma \parallel \bar{\boldsymbol{e}} \parallel \cos \theta_{Nd} \\ h_y &\triangleq k_p e_y - \eta \sigma \parallel \bar{\boldsymbol{e}} \parallel \sin \theta_{Nd} \\ k_a &> 0, \quad k_p \in (0, k_a), \quad \eta \in (0, k_p) \end{split} \qquad \begin{aligned} &\theta_{Na} \triangleq \operatorname{Atan2c} \left(\sigma \cdot h_y, \sigma \cdot h_x\right) \in \mathbb{R} \\ &\dot{\theta}_{Na} = (\dot{h}_y h_x - h_y \dot{h}_x) / (h_x^2 + h_y^2) \\ &\cos \alpha \triangleq (h_x \cos \theta_N + h_y \sin \theta_N) / \sqrt{h_x^2 + h_y^2} \end{aligned}$$

M. Michalek, K. Kozlowski: Vector-Field-Orientation feedback control method for a differentially driven vehicle. IEEE Trans. Cont. Sys. Techn., 18(1), 2010 M. Michalek: Application of the VFO method to set-point control for the N-trailer vehicle with off-axle hitching, Int. Journal of Control, 85(5), 2012

Interface subsystem



Signals conversion for backward maneuvers:

$$\beta_{0c}(\boldsymbol{u}_{0c}(\boldsymbol{e},\boldsymbol{\beta})) \triangleq \left\{ \begin{array}{ll} \operatorname{Atan2}\left(-L_0\omega_{0c}(\boldsymbol{e},\boldsymbol{\beta}),-v_{0c}(\boldsymbol{e},\boldsymbol{\beta})\right) & \text{for} & \|\boldsymbol{u}_{0c}\| > 0 \\ 0 & \text{for} & \|\boldsymbol{u}_{0c}\| = 0 \end{array} \right.$$

where

$$m{u}_{0c}(m{e},m{eta}) = [\omega_0(m{e},m{eta}) \quad v_0(m{e},m{eta})]^ op$$
 (control suggestion for DDV kinematics)

Note: Driving velocity $v_{\cal F}<0$ can be freely governed by a driver with a speed-pedal

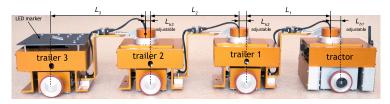


- 1 Introduction
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Experimental testbed – selected details

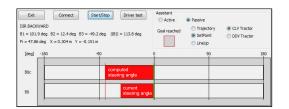
Laboratory-scale N-trailer vehicle

Introduction



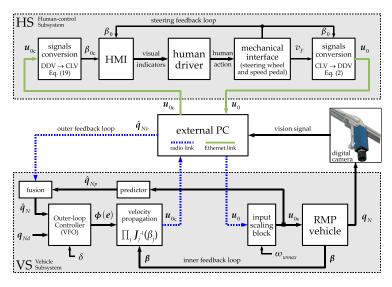
Selected for experiments: $L_{1,2,3}=0.229\,\mathrm{m},$ $L_{h1,2,3}=0.048\,\mathrm{m},$ $L_0=0.17\,\mathrm{m}$ (for emulation of CLV tractor)

Human-Machine Interface (minimalist HMI)



Only single suggested variable β_{0c} has to be tracked by a driver (steering angle β_{0c} determines motion curvature; driving velocity is secondary)

Experimental testbed – implementation scheme



Introduction

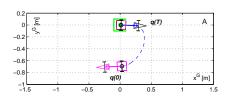
Conclusions

Control-assistance for N-trailer vehicles

Selected parameters

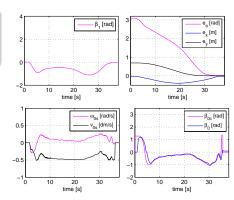
Introduction

$$\begin{array}{l} k_a = 2,\, k_p = 1,\, \eta = 0.8,\, \gamma = 0.4,\, \sigma = -1\\ w = 0.001,\, \delta = 0.02\\ \boldsymbol{q}_{Nd} = \boldsymbol{0} \end{array}$$

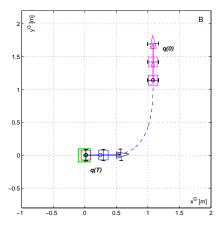


q(0) – initial vehicle configuration

q(T) – terminal vehicle configuration



Experiment B: perpendicular docking with 2-trailer (N=2)

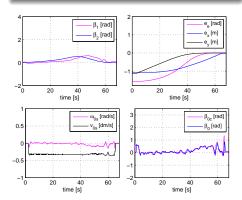


- q(0) initial vehicle configuration
- q(T) terminal vehicle configuration

Selected parameters

$$k_a = 2, k_p = 1, \eta = 0.8, \gamma = 0.4, \sigma = -1$$

 $w = 0.001, \delta = 0.02$
 $\mathbf{q}_{Nd} = \mathbf{0}$

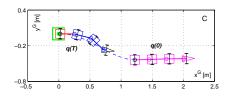


Experiment C: parallel docking with 3-trailer (N=3)

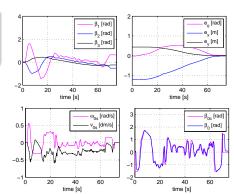
Selected parameters

$$k_a = 2$$
, $k_p = 1$, $\eta = 0.6$, $\gamma = 0.4$, $\sigma = -1$
 $w = 0.001$, $\delta = 0.02$

$$q_{Nd} = 0$$



- q(0) initial vehicle configuration
- q(T) terminal vehicle configuration





- 1 Introduction
- 2 Vehicle kinematics and control objective
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Summary of control-assistant properties

Introduction

- Highly scalable control law (arbitrary number of trailers admissible)
- Outer-loop controller determines the task (modular approach)
- Implementation and tuning simplicity (low-cost embedded solution)
- Minimalist HMI complies perception limitations (single bar-like indicator)



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Thank you for attention

